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## Shut-off fitting\_\_\_\_

The invention relates to a shut-off instrument, as described in the preamble of claim 1.

From DE 19 16 347 A1 a method for producing a shut-off instrument and a shut-off instrument are known, which comprises a one-piece instrument housing with a housing part forming a flow channel and a housing extension which forms a slide valve chamber for a wedge-type valve and a stem bushing with sealing arrangement and is joined in one piece to the housing part, for completely mounting a shut-off element designed with the slide valve stem to be movable as a wedge-type valve, which is guided in the instrument housing in such a way that it cannot be rotated. The design relates to a so-called hard-sealing slide valve instrument and sealing elements forming opposite sealing surfaces surrounding the flow channel in the instrument housing are arranged, to which compatible sealing faces on the wedge-type valve are allocated. In order to manufacture a one piece housing for a shut-off instrument of this kind an expensive mould structure and moulding procedure are necessary, to position the premanufactured wedge-type valve into a mould core and position it with the latter into a casting mould.

Furthermore, a shut-off instrument with a housing that is essentially in one piece is also known from DE 33 02 979 A1, which is provided with a lateral assembly opening for inserting the shut-off element and a bearing arrangement, and which can be sealed with a housing lid. In order to achieve the required level of tightness, in particular at higher nominal pressures, it is necessary to machine the housing and the housing lid on the sealing surfaces at great expense and to design the securing of the housing lid in a particular way.

The objective of the invention is to create a shut-off instrument independently of the design as a one-piece or multipart housing, by means of the entire pressure range can be controlled, and due to the simple assembly of the elements of the instrument made from materials adapted to the properties of the medium to be controlled, the shut-off instrument is to be used universally.

The said objective of the invention is achieved by the features described in the characterising part of claim 1. The advantage of the design according to the invention is that the components forming the shut-off element, comprising a main body, sealing elements and if required support and securing means, are inserted through the flow channel in operative position into the instrument housing, the main body can be coupled with the stem nut and the shut-off element can be completed in assembly stages, without expensive assembly aids being necessary.

A design according to claim 2 is also advantageous however, because thereby the instrument housing is manufactured in such a way that the components for the seal and bearing arrangement can be assembled in the region of the slide valve stem bushing without remachining.

A design according to claim 3 is advantageous, as thereby the bearing of the prefabricated components needed for the final assembly is simplified due to the low number of parts, and due to the short assembly time the instrument can be assembled only to fulfil the orders actually received.

A design according to claim 4 is also advantageous, as thereby simplified production methods can be applied for manufacturing the instrument housing.

According to the advantageous designs described in claims 5 to 8 it is possible to provide in each case a housing division suitable for a specific purpose of the shut-off instrument.

According to the advantageous development described in claim 9 the housing parts of multipart housings can be sufficiently and reliably joined together even for high-pressure ranges.

According to the advantageous designs described in claims 10 to 13 a torque applied by the adjustment of the shut-off element by the slide valve stem is taken up by the guides of the stem nuts, and thereby the shut-off element, in particular the sealing elements of the shut-off element, is protected effectively from asymmetrical loading and thereby uneven wear and damage.

According to the advantageous developments described in claims 14 and 15, a simple coupling procedure is achieved for producing the moving connection between the stem nut and the main body of the shut-off element during assembly, in particular in a design of the shut-off element with a one-piece instrument housing.

According to the advantageous design described in claim 16, a component that is simple to manufacture is obtained for the main body of the shut-off element.

Advantageous designs are also described in claims 17 to 19, whereby particularly for greater nominal widths, such as for example over 100 mm, the structure of the shut-off element is simplified and the assembly within a one-piece housing is facilitated.

Further advantageous designs are described in claims 20 to 24, whereby the pressure acting on the shut-off element is absorbed by the medium in the guiding arrangement and kept apart from the slide valve stem and the smooth adjustment of the shut-off element is ensured. The arrangement of guide elements in the main body, which project over the external dimensions of the latter in the region of the middle axis, is particularly advantageous, because thereby there is a symmetrical removal of pressure into the guiding arrangement.

A one-piece structure of the main body with the guiding elements is however also possible, as described in claim 25, because in this way the assembly is simplified.

A further advantageous design is described in claim 26, because thereby the completion of the shut-off element is simplified by assembly of the sealing elements in the coupled state of the main body with the stem nut, i.e. in the region of the instrument housing.

According to the advantageous development described in claims 27 to 29 the leakage of medium through the shut-off element is effectively prevented in the shut position.

Advantageous developments are also described in claims 30 to 33, as in this way a reliable and easily assembled sealing arrangement is achieved in the region of the slide valve stem bushing even at the highest pressures.

A design according to claim 34 is also advantageous, in which additional securing elements can be dispensed with and assembly is simplified.

An advantageous development according to claim 35 prevents rotation and thereby prevents the unintentional detachment of the seal carrier from the housing neck extension.

According to the advantageous designs described in claims 36 to 40, a particularly smooth bearing of the slide valve stem taking up radial and axial loads is achieved, whereby the smoothness of the adjustment is ensured.

A design described in claim 41 is also possible, as thereby the bearing elements can be dispensed with, e.g. by sealing a bearing gap with a hardening plastic composition suitable for bearing purposes.

According to an advantageous design as described in claim 42 a reliable, sealing housing neck seal is achieved, which is simple to assemble and disassemble, whereby the maintenance work and replacement of the seals or bearing elements are simplified.

According to claims 43 to 49 an advantageous design is described, in which a versatile shut-off instrument is made possible by using a combination of various different materials for the individual elements of the shut-off instrument and the instrument housing, whereby the shut-off instrument can be used in low pressure pressure conditions up to high pressure conditions, and for media of all kinds including highly aggressive media and within a broad temperature range.

The advantageous design described in claim 50 ensures pressure equalisation and a reliable seal by preventing excess pressure on the seal elements to obtain even pressure on the sealing surfaces, whereby premature wear or damage to the seal elements is effectively prevented.

Finally, a design described in claim 51 is also advantageous, whereby a multiple sealing effect and greater elasticity of the seal elements are achieved in the bearing region, and

thereby fewer elastic materials need to be used, which are advantageous for many purposes, and a greater service life can be obtained.

For a better understanding of the invention the latter is explained in more detail by way of the embodiments illustrated in the figures. Of these:

- Fig. 1 shows a shut-off instrument in cross section along the line I-I of Fig. 2;
  Fig. 2 shows the shut-off instrument, in cross section along the line II-II of Fig. 1;
  Fig. 3 shows the shut-off instrument, in cross section along the line III-III of Fig. 1;
  Fig. 4 shows a different embodiment of the shut-off instrument in cross section;
  Fig. 5 shows the shut-off instrument according to the invention with a divided housing, in simplified view;
  Fig. 6 shows a further design of a divided instrument housing in simplified view;

shows a further design of a divided instrument housing in simplified view;

Fig. 7

- Fig. 8 shows a variant of a coupling arrangement in detail with the shut-off element and the stem nut, partially in cross section;
- Fig. 9 shows the coupling arrangement in cross section along the line IX-IX of Fig. 8;
- Fig. 10 shows a further design of the instrument housing with a guiding arrangement for the shut-off element in plan view of a housing half section;
- Fig. 11 shows a further design of the shut-off instrument according to the invention in cross section along the line XI-XI of Fig. 12;

- Fig. 12 shows the shut-off instrument in cross section along the line XII-XII of Fig. 11;
- Fig. 13 shows a further design of the shut-off element according to the invention in cross section;
- Fig. 14 shows a detail of the shut-off element in cross section along the line XIV-XIV of Fig. 13.

Firstly, it should be noted that in the variously described embodiments the same parts are allocated with the same reference numbers or the same component names, whereby the disclosures contained throughout the description can be applied analogously to the same parts with the same reference numbers or the same component names. In addition, the details relating to position such as e.g. top, bottom, side etc. relate to the Figure actually being described and when there is a change in position should be transposed to the new position. Furthermore, individual features or combinations of features from the shown and described various embodiments can in themselves represent independent, inventive solutions according to the invention.

In Figs. 1 and 2 a shut-off instrument 1, in particular a slide valve instrument 2 is shown. The latter comprises an instrument housing 3 which surrounds a flow channel 6 for a medium – arrow 7- between spaced apart connection elements 4, in the shown embodiment connection flanges 5. The flow cross section 8 is a nominal width 9 of the flow channel 6. Furthermore, the instrument housing 3 with a housing projection 10 forms a slide valve chamber 11 and in an adjoining housing neck extension 12 forms a stem bushing 13 with a seal arrangement 14 and a bearing arrangement 15 for a slide valve stem 16. The slide valve stem 16 projects with a stem projection 17 through the housing neck extension 12 into the slide valve chamber 11 and in the region of the slide valve chamber 11 comprises an external thread 18 with which a stem nut 19 with an internal thread 21 arranged inside a bore 20 is engaged. The stem nut 19 is detachably connected via a detachable coupling arrangement 22 to a shut-off element 23.

A longitudinal middle axis 24 of the housing projection 10, housing neck extension 12 and the slide valve stem 16 is perpendicular to a middle axis 25 of the flow channel 6.

The slide valve chamber 11 formed by the housing projection 10 has a length 26 in the direction of a longitudinal extension which is designed for the complete clearance of the flow cross section 8 by the shut-off element 23, and corresponds roughly to the nominal width 9 in addition to the height 27 of the stem nut 19.

The shut-off element 23 is in the form of a disc-shaped main body 28 with an external dimension 29 which is slightly less than the nominal width 9. Facing the stem nut 19 the main body 28 comprises coupling grooves 30 for the engagement of hoe-like coupling means 31 of the stem nut 19, whereby the detachable coupling arrangement 22 is formed between the stem nut 19 and the main body 28 of the shut-off element 23.

Opposite bearing surfaces 32 of the main body 28 have a structure in the form of ribs 33 and on said bearing surfaces 32, disc-like so-called soft sealing elements 34 are arranged with a compatibly designed structure, to form an interlocking arrangement. Said sealing elements 34 have a circumferential edge collar 35, whereby the external dimension 36 is greater than the nominal width 9, and are provided for the sealing arrangement on opposite sealing surfaces 38 formed by a housing shaping 37.

In order to support the sealing elements 34 and secure them to the main body 28 support discs 40 arranged on opposite end faces 39 of the seal elements 34 are provided, which are secured by securing means 41, e.g. thread bolts 43 and thread nuts 44 traversing the main body 28, the sealing elements 34 and the support sheets 40 in bores 42.

It should also be mentioned that the main body 28 concentric to the longitudinal middle axis 24 is provided with a mounting bore 45 extending over the entire diameter 29 for feeding through the slide valve stem 16, whereby the internal diameter 46 is slightly greater than the thread external diameter 47 of the slide valve stem 16.

The flow channel is designed to have preferably a circular flow cross section 8 and on both

sides of the shut-off element 23 flow channel sections are in alignment with one another and have an even flow cross section 8. In a design of this kind the nominal width 9 corresponds to a nominal diameter of the flow channel 6 and the external dimension 29 of the main body 28 corresponds to an external diameter.

The flow channel 6 can also have a different geometric shape, such as for example an oval or elliptical shape.

According to the invention it is essential for the external dimension 29 of the main body 28, with the round shape diameter, to be slightly smaller than the nominal width 9 corresponding to a nominal diameter of the flow channel 6.

With an oval or elliptical shape it is essential according to the invention to make the smallest external dimension 29 of the main body slightly smaller than the minimal size of the nominal width 9 of the flow cross section 8.

Said allowances considerably simplify the insertion of the main body 28, coupling with the stem nut 19 and assembly of the seal elements 34 in a one-piece instrument housing 3, whereby assembly aids can be dispensed with and assembly time is saved.

In the region of the opposite housing shapings 37 in the centre of a middle distance 48 between the sealing surfaces 38, guide webs 49 projecting in the direction of the main body 28 are formed with a roughly triangular cross section, which run up to the region of the slide valve chamber 11, which cooperate with guide elements 50 fixed in the main body 28 and projecting over its diameter 29 in the direction of the guide webs 49, whereby a linear guide arrangement 51 is obtained as well as means for preventing the twisting of the stop element 23. A clear space 52 between the guide webs 49 is equal to or slightly greater than the nominal width 9. The guide elements 50 are arranged for example in slot-shaped recesses 53 of the main body 28 and fixed in the recess 53 by securing means 41 traversing the main body 28. Preferably, the guide elements 50 are formed by pairs of guide sheets 54, which are angled in the projecting section, which surround the guide webs 49 in a V-shape.

The shut-off element 23 is activated, as already described above, by means of the stem nut 19 coupled by the coupling arrangement 22 to the shut-off element 23, the stem nut engaging with the external tread 18 of the slide valve stem 16. Furthermore, the stem nut 19 is guided adjustably unable to be twisted in the slide valve chamber 11 in the direction of movement according to double arrow 55. In addition, guides 56 are arranged in the housing projection 10 which are diametrically opposite in relation to the longitudinal middle axis 24 and run parallel to the longitudinal middle axis 24, in which guides the guide extensions of the stem nut 19 engage. In this way a linear movement of the stem nut 19 and thereby the shut-off element 23 is achieved on the basis of a rotational movement of the slide valve stem 16 rotatably mounted in the housing neck extension 12.

In the housing neck extension 12 in the bearing arrangement 15 the slide valve stem 16 is rotatably mounted by a peripheral bearing collar 57 in a bearing sleeve 59 arranged in the housing neck top 12 and provided with a bearing flange 58. On the upper side of the bearing collar 57 opposite the bearing flange 58 a further face seal ring 60 is arranged, which is positioned against the bearing collar 57 by means of a seal carrier 61 inserted in the housing neck extension 12 into the stem bushing 13 and surrounding the stem projection 17. This design of the bearing arrangement 15 ensures the play-free, radial and axial bearing of the slide valve stem 16 in the housing neck extension 12.

The seal carrier 61 comprises an O-ring 64 in a bore 32 for the stem projection 17 in at least one peripheral groove 63, whereby preferably a further O-ring is arranged in a groove spaced further apart to decrease the likelihood of leakage. Furthermore, the seal arrangement 14 comprises a seal to prevent the leakage of the medium at a certain pressure level, which is preferably in the form of a pair of O-rings 65 arranged between an external circumference 66 of the seal carrier 61 and a mounting bore 67 of the housing neck extension 12. A bearing bore 69 facing the seal carrier 61 mounting the inner surface 68 of the mounting bore 67 as well as the bearing sleeve 59 runs from an end face 70 of the housing neck extension 12 in the direction of the slide valve chamber 11 corresponding to a surface of a conical casing, the lines of which run together in the direction of the slide valve chamber 11. The seal carrier 61 is also designed to have a truncated-cone shaped surface at least in the region of a seal projection.

In this way with a one-piece design of the instrument housing 3 a cast design with the lowest manufacturing tolerances is achieved, by means of which subsequent, expensive manufacturing procedures, such as e.g. a metal-removing rotatory procedure in the region of the stem bushing 13 becomes unnecessary, which results in considerable economic savings, and also means that the parts are of the highest quality in terms of seal tightness and replaceability.

For the axial securing of the seal carrier 61 a neck ring 73 is provided surrounding the stem projection 17 in an annular recess 71 of the housing neck extension 12 and the seal carrier 61 with support surfaces 72, which neck ring is locked e.g. by a locking arrangement 74 formed by hoe-shaped spring arms 75 in locking mounts 76 of the housing neck extension 12.

A preferred design of the seal carrier 61 and the mount 68 is illustrated in Fig. 3. According to this design the seal carrier 61 comprises locking elements 77 which are diametrically opposite one another in relation to the longitudinal middle axis 24 and project beyond the external circumference 66. The mount 68 for the seal carrier 61 comprises mounting grooves 78 for mounting the locking elements 77 which face the latter in a locked position. The locking elements 77 and the mounting grooves 78 extend respectively over a quarter of the circular circumference of the external circumference 66 of the seal carrier 61 and the mount 68 in the housing neck extension 12. In this way a bayonet type of locking of the seal carrier 61 in the housing neck extension 12 is achieved and a position is reached at which the O-rings 65 of the seal arrangement 14 are preloaded due to the conicity of the mount 68.

The seal carrier 61 is shown in its locked position in the housing neck extension 12. In order to insert the seal carrier 61 during assembly or to remove it, e.g. for maintenance work, in particular when replacing the O-rings, the seal carrier is rotated from the shown position into a position pivoted about 90°, whereby the locking elements 77 are disengaged from the mounting grooves 78 and the seal carrier 61 can be removed from the mounting bores 67. The installation is performed in the opposite manner.

In Fig. 4 the shut-off instrument 1 is shown with the shut-off element 23 with an open flow channel 6, whereby the shut-off element 23 is positioned completely inside the slide valve chamber 11. In this case, the slide valve stem 16 in the region of the external thread 18 penetrates the main body 28 of the shut-off element 23 completely after a linear movement of the stem nut 19 according to arrow 79, when the slide valve stem 16 is opened at the stem projection 17. This kind of activation can be achieved manually by a hand wheel 80 engaging directly on the stem projection 17 but in the case of an automated shut-off instrument 1 by means of a motor drive, e.g. electric drive, compressed air drive etc.

In order to restrict the adjustment a stop arrangement 81 for example is provided between the slide valve stem 16 and the shut-off element 23, e.g. a detent disc 83 positioned at an end region 82 of the slide valve stem 16, said detent disc being fixed onto the slide valve stem 16 by means of a securing nut 84 and forming a stop plane 85 perpendicular to the longitudinal middle axis 24 for a lower side 86 of the main body 28. In this way a reliable end stop is obtained in the closed position of the shut-off element 23.

Furthermore, in Fig. 4 a possible variant of the design of the instrument housing 3 is shown by dashed lines. According to this variant the housing neck extension 12 forming the stem bushing 13 with the seal arrangement 14 and the bearing arrangement 15 is secured to the housing projection 10 forming the slide valve chamber 11 to obtain a multipart housing by means of a flange arrangement 87.

The design of the shut-off instrument 1 according to the invention makes it possible, with the option of using various different materials for the instrument housing 3, shut-off element 23 with main body 28, support discs 40, seal elements 34 as well as the slide valve stem 16 and the seal carrier 61, the seal arrangement 14 and the bearing arrangements 15, to use it for media for all kinds even highly aggressive media and from low pressures up to high pressures and for a broad temperature range, e.g. between minus 50° and plus 300° C

The shut-off instrument 1 according to the invention is also characterised by the modular structure of the shut-off element 23, whereby the assembly and manufacture of the compo-

nents is simplified, and there is an economic advantage as well as high manufacturing quality, which is particularly advantageous for serial production.

The simplification of assembly is achieved in that the components forming the shut-off element 23, such as the main body 28, sealing elements 34, support discs 40 and the required securing means 41 are fed through the flow channel 6 and can be assembled without expensive assembly aids. This also makes maintenance or servicing simple as it is possible to replace components in a straight forward manner.

In Figs. 5 to 7 division possibilities are shown in a simplified form for forming a multipart instrument housing 3. In order to connect the housing parts forming the instrument housing 3, to simplify the drawing methods know from the prior art, such as connecting flanges etc., are not shown.

In Fig. 6 a division of the instrument housing 3 is shown in a division plane 91, in which the longitudinal middle axis 24 of the housing projection 10 runs and is aligned perpendicular to the middle axis 25 of the flow channel 6.

In Fig. 5 in order to form a multipart shut-off element 3 a division is shown in a division plane 90, in which the longitudinal middle axis 24 of the housing projection 10 and the middle axis 25 of the flow channel 6 run together.

In Fig. 7 a division of the instrument housing 3 is shown, whereby the middle axis 25 of the flow channel 6 runs in this division plane 92 and is aligned perpendicular to the longitudinal middle axis 24 of the housing projection 10.

In Figs. 8 and 9 a different design of the coupling arrangement 22 between the main body 28 and the stem nut 19 is shown. According to this design the stem nut 19 comprises a peripheral collar 94 at an end section 93 facing the main body 28. The main body 28 is provided with a coupling mount 96 running over the entire thickness 95, which by its design overlaps the collar 94 in parts, and thereby a moving connection is obtained between the stem nut 19 and the main body 28 for adjusting the main body 28 along the longitudinal

middle axis 24 of the slide valve stem 16, without a torque being transmitted from the stem nut 19 to the main body 28.

In Fig. 10 a further embodiment of the guide arrangement 51 in the instrument housing 3 for the shut-off element 23 is shown, whereby in the illustration only a half section is shown. According to this design in the housing shaping 37 there are guide grooves 96 which run parallel and diametrically opposite the longitudinal middle axis 24, roughly in the middle between the sealing surfaces 38. In the main body 28 guide elements 97 are secured projecting over the latter laterally and reaching into the guide grooves 96. Preferably, the guide elements 97 are made from more than one piece and comprise a guide support element 98 also secured in the main body 28 by the securing means 41 for the seal elements 34 and support discs 40, which guide support element is for diverting the force, and also comprises a slide element 99 secured to the latter, e.g. made of a low friction material, in order to achieve a smooth adjustment of the shut-off element 23 in the guide arrangement 51.

Returning to Fig. 2 it should also be mentioned that a further advantageous design of the shut-off instrument 1 according to the invention is to divide the edge collars 35 of the seal elements 34 provided on the sealing surfaces 38 in the region of the housing shaping 37 for providing a seal by a groove running concentric to the external dimension 36 and thereby form sealing lips 100, 101, whereby a better sealing effect is achieved overall, but there is also the advantage that for the seal elements 34 a harder more resistant material with a correspondingly long lifetime can be used.

In Figs. 11 and 12 a further design of the shut-off instrument 1 is shown. According to the embodiment shown in these figures the main body 28 of the shut-off element 23 is provided in one piece with the guide elements 94 with which the shut-off element 23 is guided linearly in guide grooves 96 arranged in the instrument housing 3 and in the housing projection 10. Furthermore the main body 28 has slot-like mounts 102 extending over the region of the securing means 41 from the mounting bore 45 for the slide valve stem 16 and symmetrically to the longitudinal middle axis 24. Securing tabs 103 connected in one piece with the stem nut 19 project into the latter, which are provided with bores 104, are con-

nected by the securing means 41 to the main body 28 so as to be immobile. In this way a suitable connection and anchoring of the stem nut 19 to the shut-off element 23 is achieved for the assembly of the shut-off element 23 inside the instrument housing 3.

The embodiment shown also comprises rivet bolts 105 as connecting means 41 by means of which the components of the shut-off element 23, comprising the main body 28, the sealing elements 34 applied on both sides thereof, and the support discs 40 are permanently connected. Preferably, the seal elements 34 comprise bores 106 for inserting the sealing collars 107 comprising the rivet bolts 105 and facing the main body 28, which engage in compatibly matching mounts 108, which in the main body 28 contain the bore 42 for feeding through the rivet bolts 105, and produce a seal in the region of the bores 42 for the securing means.

As can also be taken from Fig. 11, in the region of the stem bushing 13 in the housing neck extension 12 the slide valve stem 16 is provided with the bearing collar 57, which in the embodiment shown is formed essentially from two annular surfaces 110 running towards one another at an angle to the longitudinal middle axis 24 in the form of truncated-cone sleeve surfaces and projecting over a stem diameter 109. The thereby roughly V-shaped guide cross section of the bearing flange 57 ensures, in connection with the bearing sleeve 59 arranged in the stem bushing 13 and the face seal ring 60, which are preferably made of a highly resistant plastic with low sliding friction for bearing purposes, a precise and long lasting bearing of the slide valve stem 16. The shaping of the bearing flange 57 together with the matching profiling of the bearing sleeve 59 and the face seal ring 60 produces a perfect radial and also axial bearing of the slide valve stem 16 in the region of the stem bushing 13.

The bearing sleeve 59 facing the shut-off element 23 projects with an annular extension 111 over a ring-shaped end face 112 facing the slide valve chamber 11 in the direction of the shut-off element 23 and thus forms a stop surface 113 for a ring-shaped end face 114 of the stem nut 19 facing the latter, whereby the movement of the shut-off element 23 into the open position, in which the flow cross section 8 is completely open, is restricted.

The seal carrier 61 inserted into the housing neck extension 12 by means of the bayonet lock, already described in the preceding Figures, comprises in the region of the end face 70 of the housing neck extension 12 at least two flanged noses 116 diametrically opposite the longitudinal middle axis 24 and engaging in notch recesses 115, by which the seal carrier 61 is secured against rotation in the bore 62 of the housing neck extension 11.

In Figures 13 and 14 a further design of the shut-off element 23 is shown in detail, in particular for shut-off instruments 1 with a greater nominal width, e.g. over 100 mm. In this design the main body 28 is formed essentially by two spaced apart, shaped wall discs 117, 118 for mounting the seal elements 34 and the support discs 40, whereby the wall discs 117, 118 form a hollow body 121 by means of wall sections 119 and spacing sleeves 120. Preferably, the wall discs 117, 118 together with the wall sections 119 and spacer sleeves 120 and the laterally projecting guide elements 97 form a one-piece component.

At an end section 122 opposite the coupling arrangement 22 between the stem nut 19 and the main body 28 the main body 28 is provided with a flattened section 123, whereby the height 124 is smaller than the diameter 125 of a theoretical circle outline of the outer contour of the main body 28.

By means of this flattened section 123 the insertion of the main body 28 into the instrument housing 3 and in particular the insertion of the guide elements 97 into the guide grooves 96 is made much easier during the assembly of the shut-off element 23 inside the instrument housing 3, whereby titling about the longitudinal axis 25 is necessary.

In order to ensure the required bearing and support for the seal elements 34 in the region of the main body 28 restricted by the flattened section 123, this section is installed before the assembly of the seal elements 34 and support discs 40, by means of a compensating element 127, completing the circular outline of the main body 28 which, can be coupled by a plug-and-socket connection 126 to the main body 28, in particular to the wall sections 119.

For form's sake it is pointed out that for a better understanding of the structure of the shutoff instrument 1 the latter and its components are not always drawn to scale and/or have been enlarged and/or reduced in size.

The objective forming the basis of the independent solutions of the invention can be taken from the description.

Mainly the individual designs shown in detail in Figs. 1, 2, 3, 4; 5; 6; 7; 8, 9; 10; 11, 12; 13, 14 can form the subject matter of independent solutions according to the invention. The objectives and solutions relating thereto can be taken from the detailed descriptions of said figures.

## List of Reference Numbers

1	Shut-off element	31	Coupling means
2	Slide valve instrument	32	Support surface
3	Instrument housing	33	Rib
4	Connection element	34	Seal element
5	Connection flange	35	Edge collar
6	Flow channel	36	External dimension
7	Arrow	37	Housing shaping
8	Flow cross section	38	Sealing surface
9	Nominal diameter	39	End face
10	Housing extension	40	Support disc
11	Slide valve chamber	41	Securing means
12	Housing neck extension	42	Bore
13	Stem bushing	43	Threaded bolt
14	Seal arrangement	44	Threaded nut
15	Bearing arrangement	45	Mounting bore
16	Slide valve stem	46	Inner diameter
17	Stem projection	47	Thread external diameter
18	External thread	48	Distance
19	Stem nut	49	Guide web
20	Bore	50	Guide element
21	Internal thread	51	Guide arrangement
22	Coupling arrangement	52	Width
23	Stop element	53	Recess
24	Longitudinal middle axis	54	Guide sheet
25	Middle axis	55	Double arrow
26	Length	56	Guides
27	Structure height	57	Bearing collar
28	Main body	58	Bearing flange
29	External dimension	59	Bearing sleeve
30	Coupling groove	60	Face seal ring

61	Seal carrier	93	End section
62	Bore	94	Collar
63	Grooves	95	Thickness
64	O-ring	96	Guide groove
65	O-ring	97	Guide element
66	External circumference	98	Guide support element
67	Mounting bore	99	Slide element
68	Surface	100	Sealing lip
69	Bearing bore	101	Sealing lip
70	End face	102	Mount
71	Recess	103	Securing tab
72	Support surface	104	Bore
73	Neck ring	105	Groove bolt
74	Locking arrangement	106	Bore
75	Spring arms	107	Sealing collar
76	Locking mount	108	Mount
77	Locking element	109	Stem diameter
78	Mounting grooves	110	Annular surface
79	Arrow	111	Annular extension
80	Hand wheel	112	End face
81	Stop arrangement	113	Stop surface
82	End section	114	End face
83	Detent ring	115	Notch recess
84	Securing nut	116	Flanged nose
85	Stop plane	117	Wall disc
86	Lower side	118	Wall disc
87	Flange arrangement	119	Wall section
88		120	Spacer sleeves
89		121	Hollow body
90	Division plane	122	End section
91	Division plane	123	Flattened section
92	Division plane	124	Height

- 125 Diameter
- 126 Part connection
- 127 Compensating element